

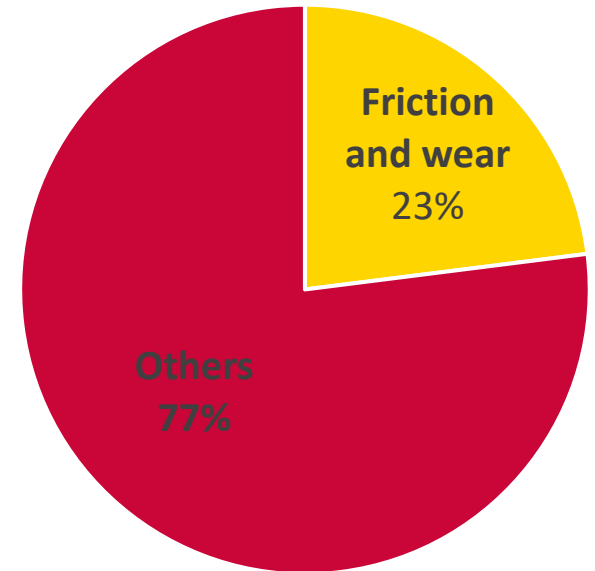
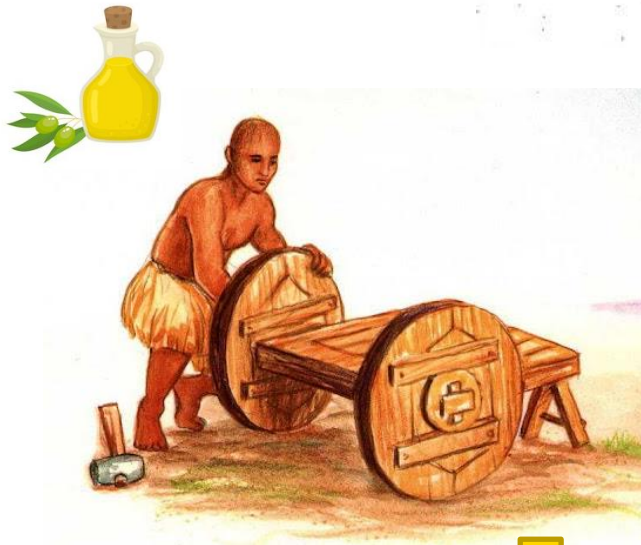
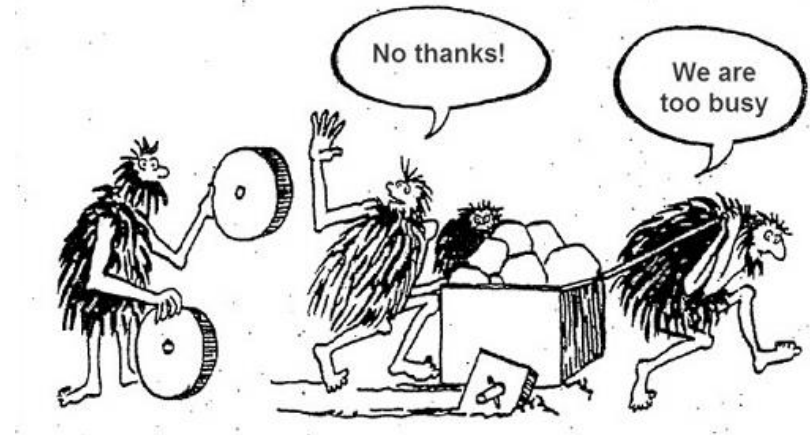
A yellow circle is positioned in the upper left quadrant of the slide. A grey line drawing of a person's head and neck is overlaid on the right side of the slide, with the head tilted back and the neck extending downwards.

«New methodology for thermal properties evaluation of energy-saving lubricating oils»

Evaluation of Lubricant Thermal Properties

Eleonora Colombo

Friction and lubrication role



Lubricants can directly act on energy saving performances



LUBRICATE

- ✓ Prevent **wear**
- ✓ Reduce **friction**

CLEAN

- ✓ Remove **contaminants**

COOL

- ✓ Transfer **heat**
- ✓ Enhance **durability**
- ✓ Prevent **corrosion**

Thermal properties of lubricants

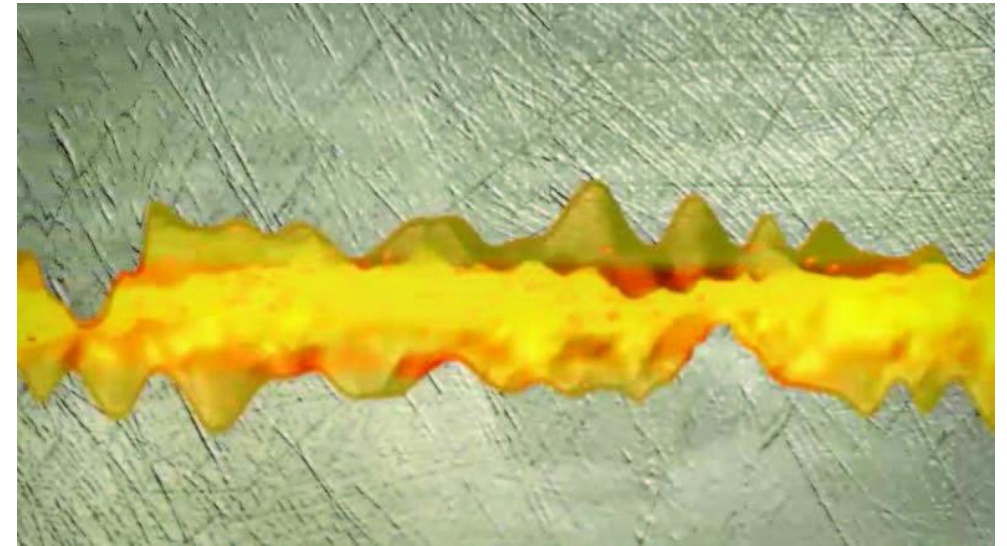
Relative motion between the mechanical parts

Friction phenomena

Heat dissipation

Lubricant thermal properties

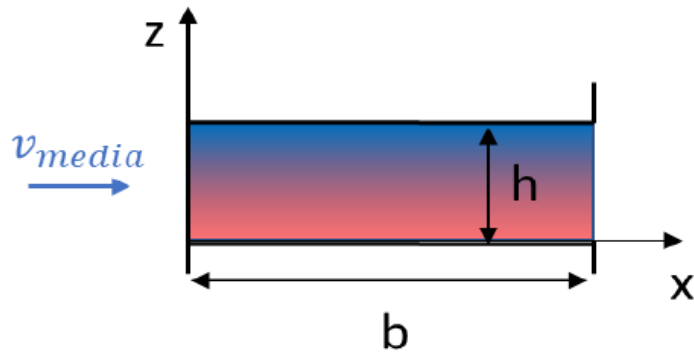
Solid surface 1



Solid surface 2

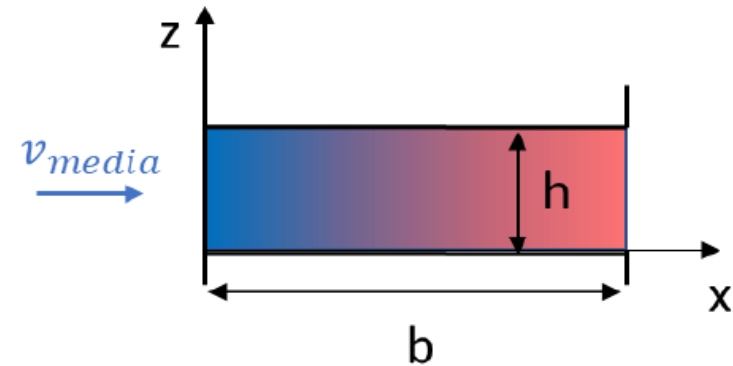
Heat transfer phenomena

Thermal conduction



Heat is transferred across the film thickness
in **z direction**

Advection



Heat is transferred along the film length
and brought away by lubricant
in **x direction**

$$Pe = \frac{\text{Advection}}{\text{Conduction}}$$

Heat transfer phenomena

$$Pe = \frac{\text{Advection}}{\text{Conduction}} = \frac{\rho c_p v_{mean}}{\lambda} \cdot \frac{h_{film}^2}{b}$$

ρc_p = product between density and specific heat

v_{mean} = mean velocity of lubricant

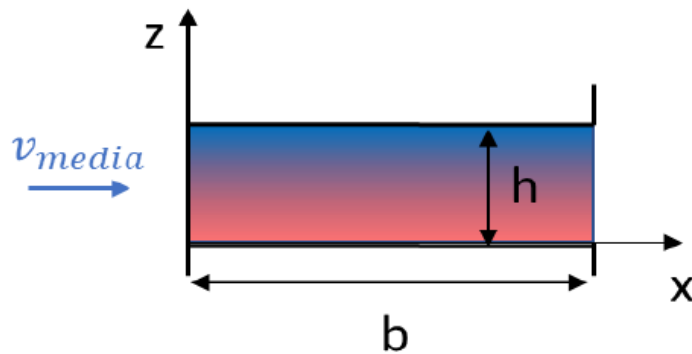
h_{film} = lubricant film thickness

λ = Thermal conductivity

b = Length of tribological coupling

$Pe \rightarrow 0$

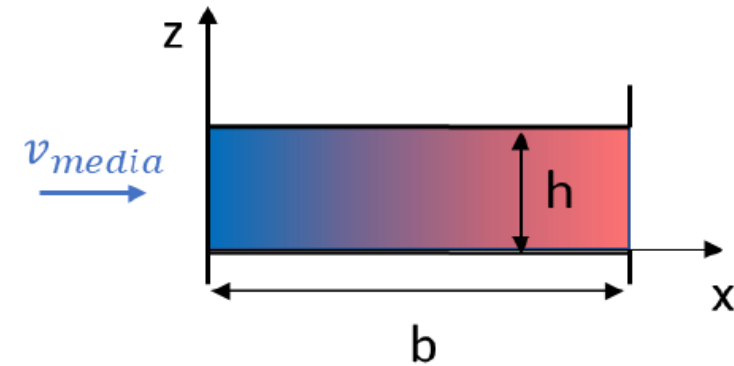
Thermal conduction



$v_{mean} \downarrow$ $h_{film} \downarrow$

$Pe \rightarrow \infty$

Advection



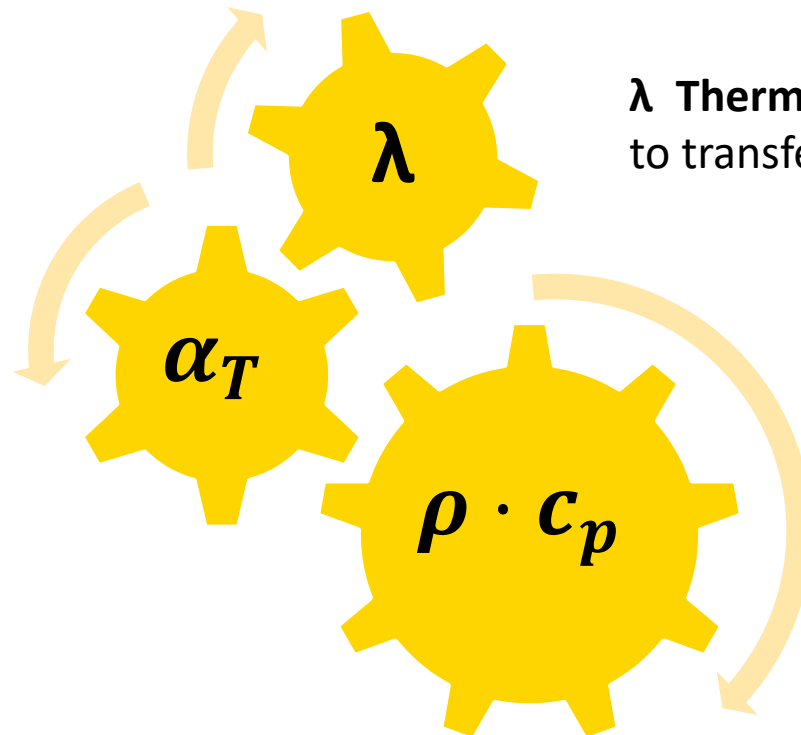
$v_{mean} \uparrow$ $h_{film} \uparrow$

Goal of the research activity

Estimate energy-saving potential of lubricating oils during lubricant design.

Analyse thermal properties of a wide range of base oils for lubricant design and of finite products

α_T **Thermal diffusivity** = rate of heat transfer in time.



λ **Thermal conductivity** = ability of a material to transfer heat through conduction.

$\rho \cdot c_p$ **Product of density and specific heat** = ability of a material to store heat and transfer it through advection.

Tested samples

Thermal properties are evaluated for different feedstocks

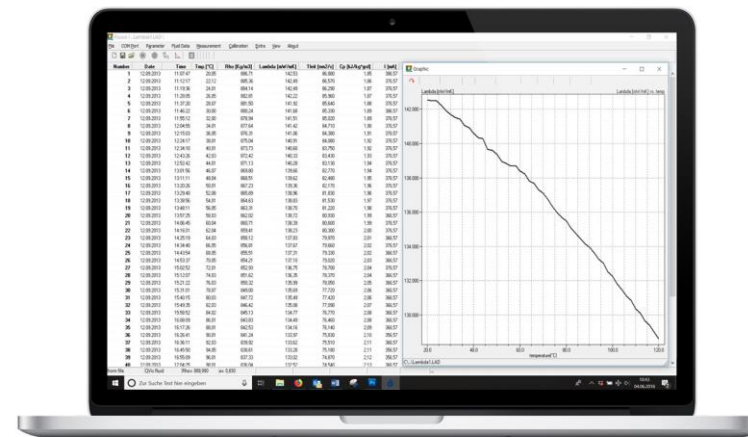
Base oil type	KV a 40°C [cSt]
PAG water insoluble	45
PAG water soluble 1	50-54
PAG water soluble 2	73-84
Renewable base oil GPIII	58
Mineral base oil GPIII	49
Re-refined base oil GPI+	58
MIX mineral base oil GPI	55.2
esters	46
MIX estolides	58.7
MIX polialphaolefins	57.8
MIX naphthenic base oil	56.7

Effects related to viscosity are minimized.

Instrumentation and principles of measurement



- *The experimental apparatus Flucon LAMBDA:*
 - *Electronic device,*
 - *Measuring probe and sample cup,*
 - *Thermostat Omega,*
 - *Laptop,*
 - *FluconLAM PC Software.*



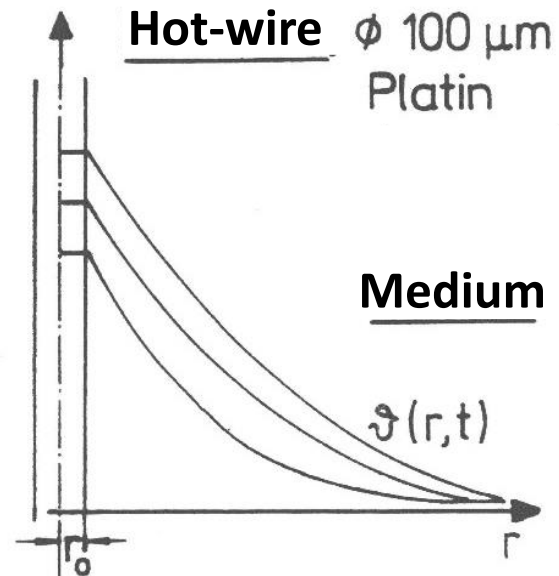
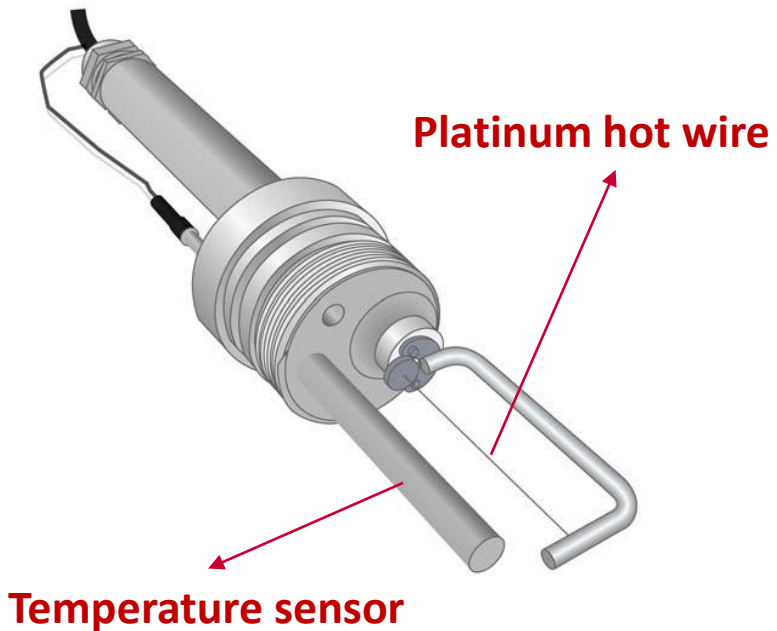
Instrumentation and principles of measurement

- Measuring principle:

Instationary hot-wire method

Thermal conductivity λ

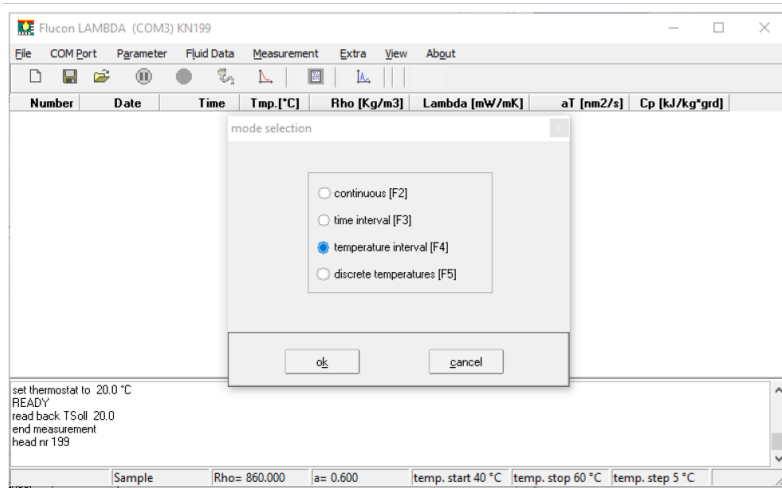
Thermal diffusivity α_T



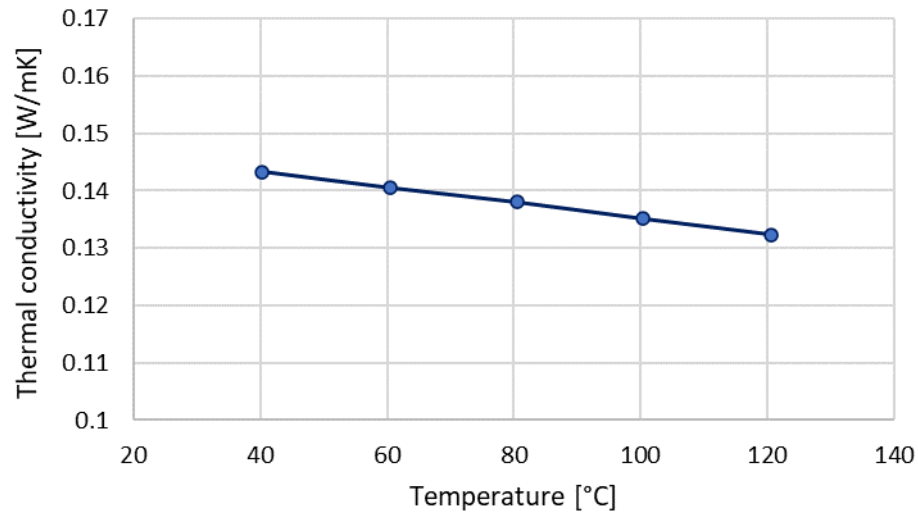
I° Fourier's law
 $q = \lambda \nabla T$

II° Fourier's law
 $\frac{\partial \vartheta}{\partial t} = \alpha_T \nabla^2 \vartheta$

Development of a new procedure



Regenerated Group 1 base oil



In one single experiment

- *Thermal properties are measured at 40, 60, 80, 100 and 120°C.*
- *5 repetitive measurement at each temperature*
- *~ 3 hours*

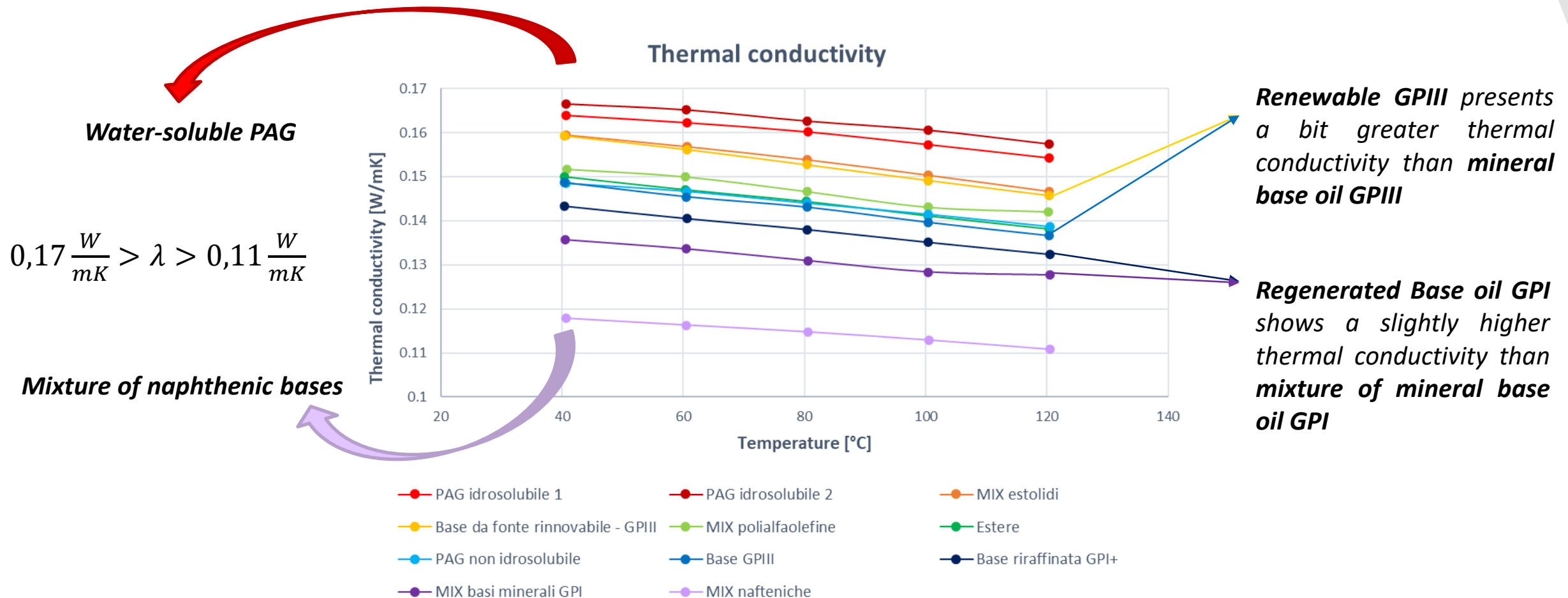


*Estimation of λ , α_T and $\rho * c_p$*

$$\rho \cdot c_p = \frac{\lambda}{\alpha_T}$$



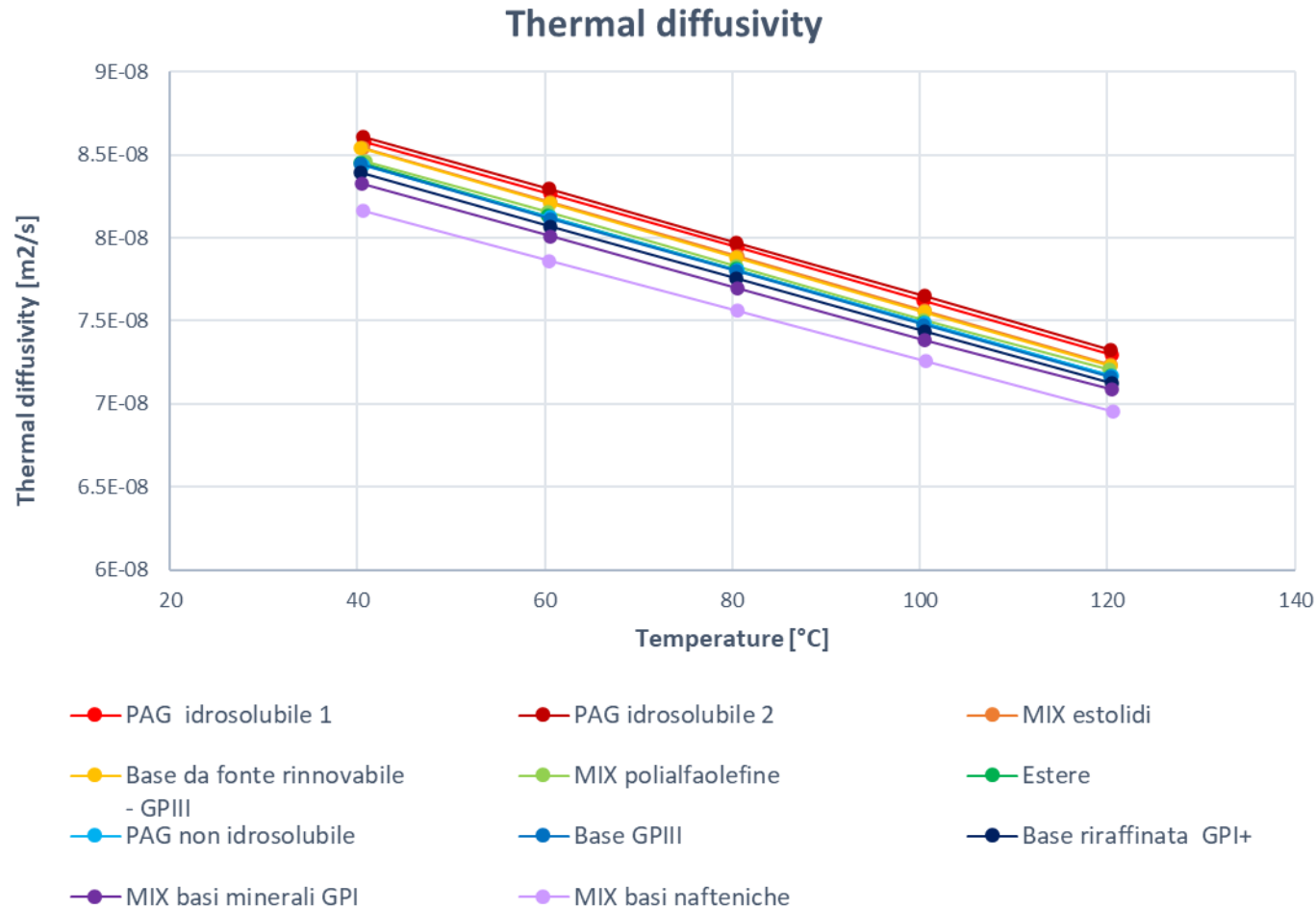
Thermal conductivity



Thermal conductivity decreases at increasing temperatures for all the analysed base oils.



Thermal diffusivity



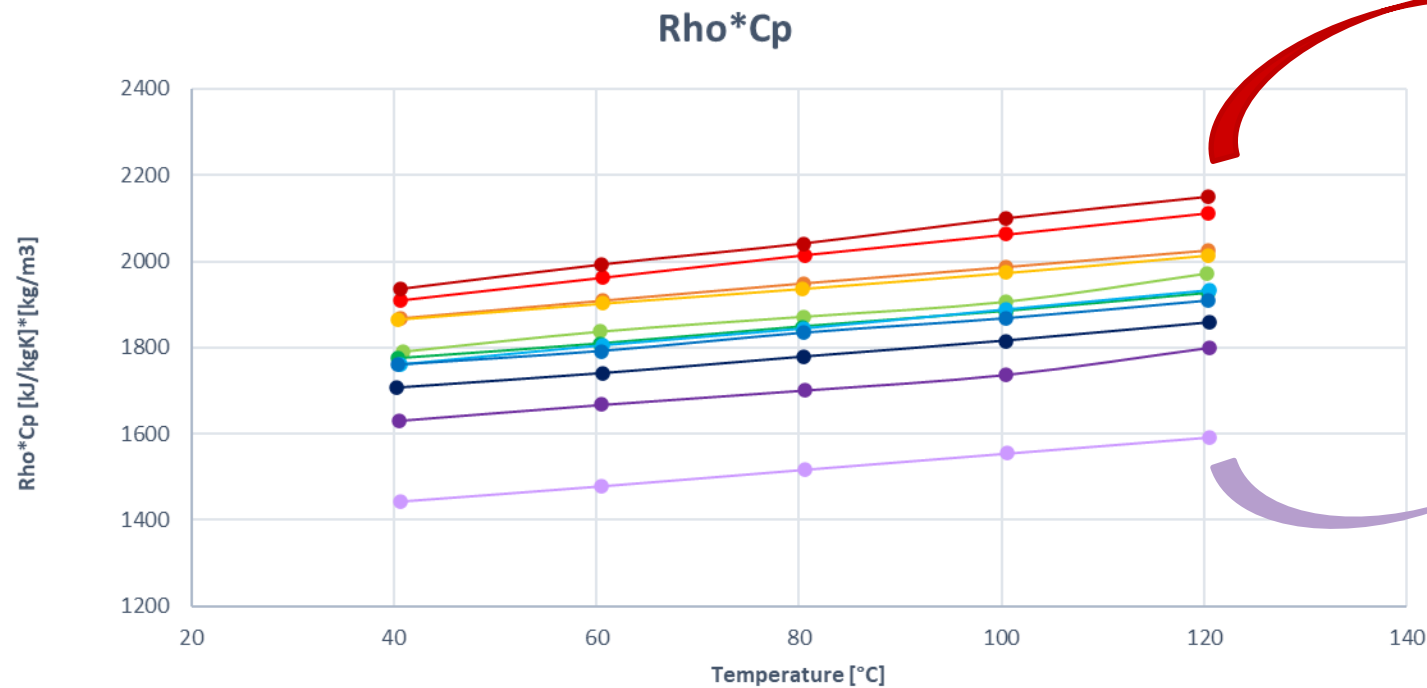
$$9 \cdot 10^{-8} \frac{m^2}{s} > \alpha > 7 \cdot 10^{-8} \frac{m^2}{s}$$

Thermal diffusivity decreases at increasing temperatures for all the analysed base oils:

The *thermal inertia* of lubricants grows when temperature grows.



Product of density and specific heat



Water-soluble PAG

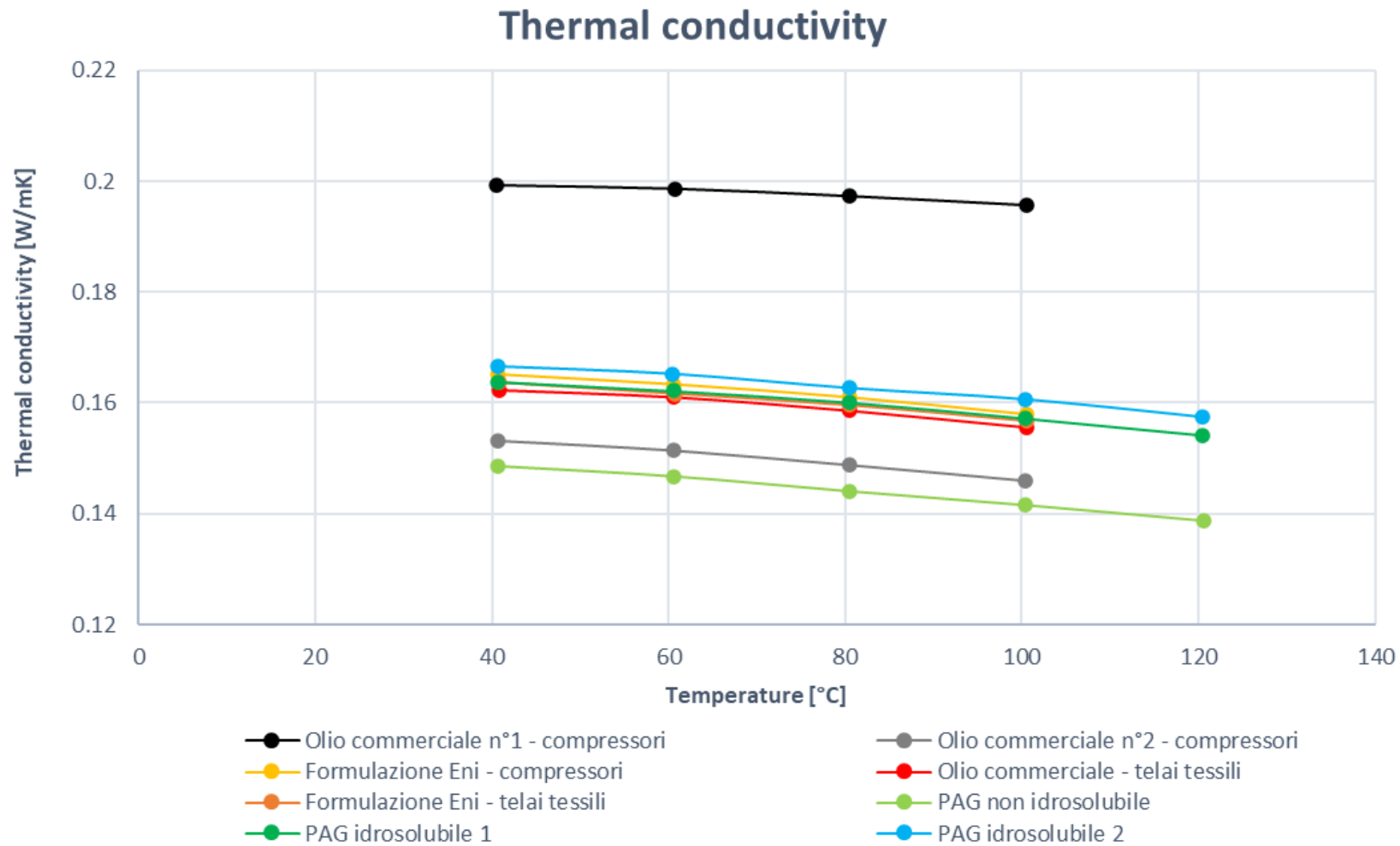
$$1400 \frac{kJ}{m^3 K} > \rho \cdot c_p > 2200 \frac{kJ}{m^3 K}$$

Mixture of naphthenic bases

The product increases at increasing temperatures for all the selected base oils.

- PAG idrosolubile 1
- PAG idrosolubile 2
- MIX estolidi
- Base da fonte rinnovabile - GPIII
- MIX polialfaolefine
- Estere
- PAG non idrosolubile
- Base GPIII
- Base riraffinata GPI+
- MIX basi minerali GPI
- MIX basi nafteniche

Comparison of thermal conductivities between base oils and finite products - PAG



The behaviour of finite lubricant is closely related to base oil behaviour



Conclusions

The developed methodology enables:

- *The evaluation of all thermal properties of lubricating oil at variable temperatures in one single experiment*
- *Provide with additional data the in-house developed products with respect to other competition companies.*
- *Gain additional information for the estimation of energy-saving performance of lubricating oils*



$$P_{diss} = \dot{V} \rho c_p * (T_{in} - T_{out})$$

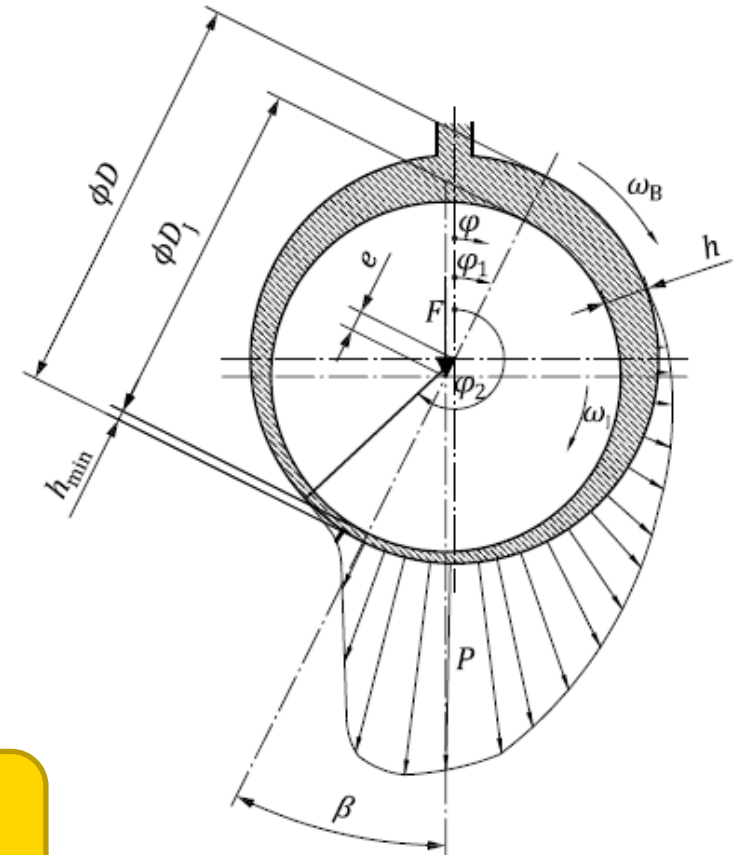


Future development

Additional experimental activity on new samples

Simple theoretical model to simulate power dissipation as a function of lubricant properties

Lubricant Design





Thank you

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 - eleonora.colombo@eni.com

References

- *A History of Lubricants. K.J. Anderson.*
- *Influence of tribology on global energy consumption, costs and emissions. K. Holmberg, A. Erdemir.*
- *Principles of tribology. S. Wen e P. Huang.*
- *Flucon, «Thermal Conductivity Meter LAMBDA Hardware Manual».*
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- *ASTM D7896 19: Standard Test Method for Thermal Conductivity, Thermal Diffusivity, and Volumetric Heat Capacity of Engine Coolants and Related Fluids by Transient Hot Wire Liquid Thermal Conductivity Method.*
- *Yaws' handbook of thermodynamic and physical properties of chemical compounds. C. L. Yaws.*
- *ISO 7902-1 2020, Hydrodynamic plain journal bearings under steady-state conditions - Circular cylindrical bearings.*

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